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## A NON-LINEAR MODEL FOR STUDYING THE MOTION OF A HUMAN BODY

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### Abstract

In our paper we developed a non-linear model for studying the motions of a human body. For our model we obtained the equations of motion using the Newton's equations. We also treated a few aspects concerning the stability of the equilibrium and motion.

## 1. INTRODUCTION

The mechanical model is captured in figure 1.

We made the following notations:  $m_i$ ,  $i = \overline{1, 4}$  - mass of the body  $i$ ;  $C_i$ ,  $i = \overline{1, 4}$  - weight center of the body  $i$ ;  $J_i$ ,  $i = \overline{3, 4}$  - inertial moment of the body  $i$  with respect to the weight center  $C_i$ ;  $h$  - distance between  $C_1$  and  $C_2$  measured in the direction of the axis  $y$ ;  $C_2C_3 = l'_3$ ;  $D_4E_4 = L_4$ ;  $C_2D_3 = l''_3$ ;  $D_4C_4 = l_4$ ;  $C_2D_4 = L_3$ ;  $F_{e_{01}}$  - elastic force in the spring between the bodies 0 and 1;  $M_{e_{23}}$  - elastic moment in the spring between the bodies 2 and 3;  $M_{e_{34}}$  - elastic moment in the spring between the bodies 3 and 4;  $F_{e_{12}}$  elastic force in the spring between the bodies 1 and 2;  $F_{e_{13}}$  elastic force in the spring between the bodies 1 and 3;  $d$  - distance between  $C_1$  and  $C_2$  measured in the direction of the axis  $x$  and for unstressed springs;  $F_f$  - coulombian friction force between the bodies 1 and 2;  $N_{01}$  - normal reaction between the bodies 0 and 1;  $N_{12}$  - normal reaction between the bodies 1 and 2. We shall consider that all elastic forces and moments are equal to zero for zero strains.

## 2. ISOLATION OF THE BODIES

For the fourth body we obtain the following equations (fig. 2):

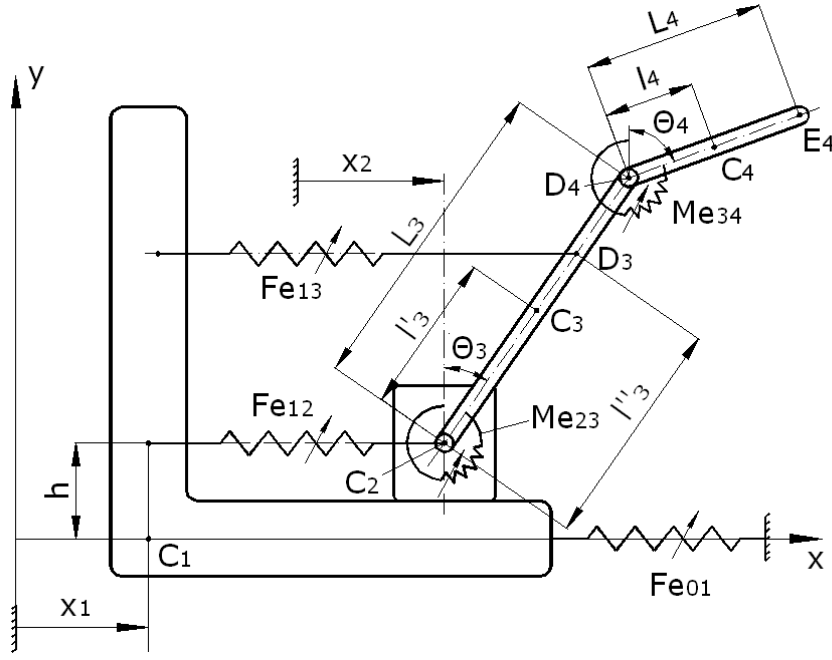


Figure 1. Mechanical model.

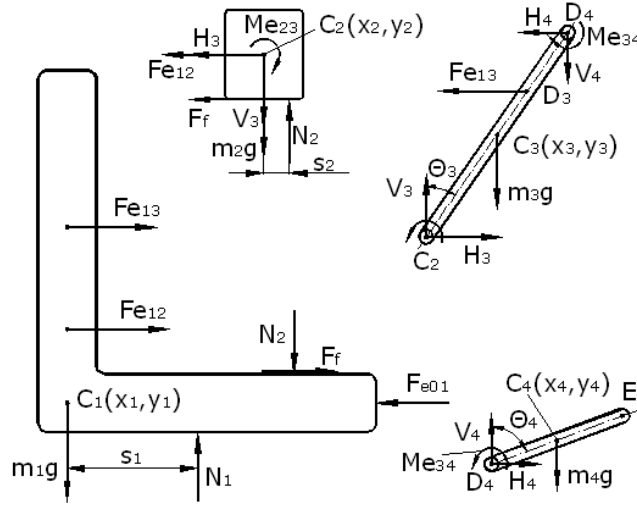


Figure 2. Isolation of the bodies.

$$m_4 \ddot{x}_4 = H_4; m_4 (\ddot{y}_4 + g) = V_4; J_4 \ddot{\theta}_4 = -H_4 l_4 \cos \theta_4 + V_4 l_4 \sin \theta_4 - M_{e34} (\theta_4 - \theta_3). \quad (1)$$

For the third body we obtain:

$$\begin{aligned} m_3 \ddot{x}_3 &= H_3 - H_4 - F_{e13} (x_2 + l_3' \sin \theta_3); m_3 (\ddot{y}_3 + g) = V_3 - V_4; \\ J_3 \ddot{\theta}_3 &= -H_3 l_3' \cos \theta_3 - H_4 (L_3 - l_3') \cos \theta_3 + V_3 l_3' \sin \theta_3 + V_4 (L_3 - l_3') \sin \theta_3 + \\ &+ M_{e34} (\theta_4 - \theta_3) - M_{e23} (\theta_3) - F_{e13} (x_2 + l_3' \sin \theta_3) (l_3'' - l_3') \cos \theta_3 = 0. \end{aligned} \quad (2)$$

For the second body and the first body it follows

$$m_2\ddot{x}_2 = -H_3 - \mu N_2 - F_{e_{12}}(x_2); 0 = N_2 - V_3 - m_2g, \quad (3)$$

respectively

$$m_1\ddot{x}_1 = \mu N_2 + F_{e_{12}}(x_2) + F_{e_{13}}(x_2 + l_3'' \sin \theta_3) - F_{e_{01}}(x_1); 0 = N_1 - m_1g - N_2. \quad (4)$$

### 3. OBTAINING THE SECOND ORDER DIFFERENTIAL EQUATION SYSTEM

It is easy to obtain the following relations:

$$J_4\ddot{\theta}_4 = -m_4\ddot{x}_4l_4 \cos \theta_4 + m_4(\ddot{y}_4 + g)l_4 \sin \theta_4 - M_{e_{34}}(\theta_4 - \theta_3) \quad (5)$$

$$\begin{aligned} J_3\ddot{\theta}_3 = & -[m_3\ddot{x}_3 + m_4\ddot{x}_4 + F_{e_{13}}(x_2 + l_3'' \sin \theta_3)]' \cos \theta_3 - m_4\ddot{x}_4(L_3 - l_3') \cos \theta_3 + \\ & + (m_3\ddot{y}_3 + m_3g + m_4\ddot{y}_4 + m_4g)l_3' \sin \theta_3 + (m_4\ddot{y}_4 + m_4g)(L_3 - l_3') \sin \theta_3 + \\ & + M_{e_{34}}(\theta_4 - \theta_3) - M_{e_{23}}(\theta_3) - F_{e_{13}}(x_2 + l_3'' \sin \theta_3)(l_3'' - l_3') \cos \theta_3 = 0, \end{aligned} \quad (6)$$

$$\begin{aligned} m_2\ddot{x}_2 = & -[m_3\ddot{x}_3 + m_4\ddot{x}_4 + F_{e_{13}}(x_2 + l_3'' \sin \theta_3)] - \\ & - \mu[m_3\ddot{y}_3 + m_4\ddot{y}_4 + (m_2 + m_3 + m_4)g] - F_{e_{12}}(x_2), \end{aligned} \quad (7)$$

$$m_1\ddot{x}_1 + m_2\ddot{x}_2 + m_3\ddot{x}_3 + m_4\ddot{x}_4 = -F_{e_{01}}(x_1). \quad (8)$$

On the other hand

$$\begin{aligned} \ddot{x}_3 = & \ddot{x}_1 + \ddot{x}_2 + l_3'\ddot{\theta}_3 \cos \theta_3 - l_3'\dot{\theta}_3^2 \sin \theta_3; \ddot{y}_3 = -l_3'\ddot{\theta}_3 \sin \theta_3 - l_3'\dot{\theta}_3^2 \cos \theta_3; \\ \ddot{x}_4 = & \ddot{x}_1 + \ddot{x}_2 + L_3\ddot{\theta}_3 \cos \theta_3 - L_3\dot{\theta}_3^2 \sin \theta_3 + l_4\ddot{\theta}_4 \cos \theta_4 - l_4\dot{\theta}_4^2 \sin \theta_4; \\ \ddot{y}_4 = & -L_3\ddot{\theta}_3 \sin \theta_3 - L_3\dot{\theta}_3^2 \cos \theta_3 - l_4\ddot{\theta}_4 \sin \theta_4 - l_4\dot{\theta}_4^2 \cos \theta_4 \end{aligned} \quad (9)$$

and replacing in the equations (5)-(8) we obtain the system

$$\begin{aligned} A_{11}\ddot{x}_1 + A_{12}\ddot{x}_2 + A_{13}\ddot{\theta}_3 + A_{14}\ddot{\theta}_4 = B_1; A_{21}\ddot{x}_1 + A_{22}\ddot{x}_2 + A_{23}\ddot{\theta}_3 + A_{24}\ddot{\theta}_4 = B_2; \\ A_{31}\ddot{x}_1 + A_{32}\ddot{x}_2 + A_{33}\ddot{\theta}_3 + A_{34}\ddot{\theta}_4 = B_3; A_{41}\ddot{x}_1 + A_{42}\ddot{x}_2 + A_{43}\ddot{\theta}_3 + A_{44}\ddot{\theta}_4 = B_4, \end{aligned} \quad (10)$$

where

$$\begin{aligned} A_{11} = & m_4l_4 \cos \theta_4; A_{12} = m_4l_4 \cos \theta_4; \\ A_{13} = & m_4L_3 \cos \theta_3 l_4 \cos \theta_4 + m_4L_3 \sin \theta_3 l_4 \sin \theta_4 = m_4L_3l_4 \cos(\theta_4 - \theta_3); \\ A_{14} = & J_4 + m_4l_4^2 \cos^2 \theta_4 + m_4l_4^2 \sin^2 \theta_4 = J_4 + m_4l_4^2, \end{aligned} \quad (11)$$

$$\begin{aligned} A_{21} = & (m_3 + m_4)l_3' \cos \theta_3 + m_4(L_3 - l_3') \cos \theta_3; \\ A_{22} = & (m_3 + m_4)l_3' \cos \theta_3 + m_4(L_3 - l_3') \cos \theta_3; A_{23} = J_3 + m_3(l_3')^2 + m_4L_3^2; \\ A_{24} = & m_4l_4l_3' \cos(\theta_4 - \theta_3) + m_4l_4(L_3 - l_3') \cos(\theta_4 - \theta_3), \end{aligned} \quad (12)$$

$$\begin{aligned}
 A_{31} &= m_3 + m_4; \quad A_{32} = m_2 + m_3 + m_4; \\
 A_{33} &= m_3 l'_3 \cos \theta_3 + m_4 L_3 \cos \theta_3 - \mu m_3 l'_3 \sin \theta_3 - \mu m_4 L_3 \sin \theta_3; \\
 A_{34} &= m_4 l_4 \cos \theta_4 - \mu m_4 l_4 \sin \theta_4,
 \end{aligned} \tag{13}$$

$$\begin{aligned}
 A_{41} &= m_1 + m_3 + m_4; \quad A_{42} = m_2 + m_3 + m_4; \quad A_{43} = m_3 l'_3 \cos \theta_3 + m_4 L_3 \cos \theta_3; \\
 A_{44} &= m_4 l_4 \cos \theta_4,
 \end{aligned} \tag{14}$$

$$\begin{aligned}
 B_1 &= m_4 L_3 l_4 \dot{\theta}_3^2 \sin(\theta_3 - \theta_4) + m_4 g l_4 \sin \theta_4 - M_{e_{34}}(\theta_3 - \theta_4); \\
 B_2 &= m_4 l_4 L_3 \dot{\theta}_4^2 \sin(\theta_4 - \theta_3) - F_{e_{13}}(x_2 + l'_3 \sin \theta_3) l''_3 \cos \theta_3 + \\
 &\quad + m_3 g l'_3 \sin \theta_3 + m_4 g L_3 \sin \theta_3 + M_{e_{34}}(\theta_4 - \theta_3) - M_{e_{23}}(\theta_3); \\
 B_3 &= m_3 l'_3 \dot{\theta}_3^2 (\sin \theta_3 - \mu \cos \theta_3) + m_4 L_3 \dot{\theta}_3^2 (\sin \theta_3 + \mu \cos \theta_3) + \\
 &\quad + m_4 l_4 \dot{\theta}_4^2 (\sin \theta_4 + \mu \cos \theta_4) - F_{e_{13}}(x_2 + l'_3 \sin \theta_3) - \\
 &\quad - \mu(m_2 + m_3 + m_4)g - F_{e_{12}}(x_2); \\
 B_4 &= m_3 l'_3 \dot{\theta}_3^2 \sin \theta_3 + m_4 L_3 \dot{\theta}_3^2 \sin \theta_3 + m_4 l_4 \dot{\theta}_4^2 \sin \theta_4 - F_{01}(x_1);
 \end{aligned} \tag{15}$$

hence  $A_{ij} = A_{ji}(x_1, x_2, \theta_3, \theta_4, \dot{x}_1, \dot{x}_2, \dot{\theta}_3, \dot{\theta}_4)$ ,  $B_i = B_i(x_1, x_2, \theta_3, \theta_4, \dot{x}_1, \dot{x}_2, \dot{\theta}_3, \dot{\theta}_4)$ ,  $i = \overline{1, 4}$ ,  $j = \overline{1, 4}$ .

The system (10) leads to

$$\begin{aligned}
 \ddot{x}_1 &= f_1(x_1, x_2, \theta_3, \theta_4, \dot{x}_1, \dot{x}_2, \dot{\theta}_3, \dot{\theta}_4); \quad \ddot{x}_2 = f_2(x_1, x_2, \theta_3, \theta_4, \dot{x}_1, \dot{x}_2, \dot{\theta}_3, \dot{\theta}_4); \\
 \ddot{\theta}_3 &= f_3(x_1, x_2, \theta_3, \theta_4, \dot{x}_1, \dot{x}_2, \dot{\theta}_3, \dot{\theta}_4); \quad \ddot{\theta}_4 = f_4(x_1, x_2, \theta_3, \theta_4, \dot{x}_1, \dot{x}_2, \dot{\theta}_3, \dot{\theta}_4).
 \end{aligned} \tag{16}$$

Denoting

$$x_1 = \xi_1; \quad x_2 = \xi_2; \quad \theta_3 = \xi_3; \quad \theta_4 = \xi_4; \quad \dot{x}_1 = \xi_5; \quad \dot{x}_2 = \xi_6; \quad \dot{\theta}_3 = \xi_7; \quad \dot{\theta}_4 = \xi_8, \tag{17}$$

it results the first order non-linear differential equations system

$$\frac{d\xi_i}{dt} = \xi_{i+4}; \quad i = \overline{1, 4}; \quad \frac{d\xi_i}{dt} = f_{i-4}(\xi_1, \dots, \xi_8); \quad i = \overline{5, 8}. \tag{18}$$

Let us observe that the system (18) is valuable for  $N_2 \geq 0$ , otherwise the system has five degrees of freedom.

Our system has four degrees of freedom :  $x_1$ ,  $x_2$ ,  $\theta_3$  and  $\theta_4$ .

#### 4. THE STUDY OF THE EQUILIBRIUM

In this case, we obtain

$$\dot{x}_1 = 0; \quad \dot{x}_2 = 0; \quad \dot{\theta}_3 = 0; \quad \dot{\theta}_4 = 0. \tag{19}$$

We study the equilibrium for the elastic forces and moments given by linear relations

$$\begin{aligned} F_{e_{12}}(x_2) &= k_{12}x_2; F_{e_{13}}(x_2 + l_3'' \sin \theta_3) = k_{13}(x_2 + l_3'' \sin \theta_3); \\ M_{e_{34}}(\theta_4 - \theta_3) &= k_{34}(\theta_4 - \theta_3); M_{e_{23}}(\theta_3) = k_{23}(\theta_3), \end{aligned} \quad (20)$$

the bars are homogeneous, and  $l_3' = \frac{L_3}{2} = l_3'', l_4 = \frac{L_4}{2}, \mu = 0$ .

It follows the system

$$\begin{aligned} A_{11}\ddot{x}_1 + A_{12}\ddot{x}_2 + A_{13}\ddot{\theta}_3 + A_{14}\ddot{\theta}_4 &= B_1; A_{21}\ddot{x}_1 + A_{22}\ddot{x}_2 + A_{23}\ddot{\theta}_3 + A_{24}\ddot{\theta}_4 = B_2; \\ A_{31}\ddot{x}_1 + A_{32}\ddot{x}_2 + A_{33}\ddot{\theta}_3 + A_{34}\ddot{\theta}_4 &= B_3; A_{41}\ddot{x}_1 + A_{42}\ddot{x}_2 + A_{43}\ddot{\theta}_3 + A_{44}\ddot{\theta}_4 = B_4. \end{aligned} \quad (21)$$

Subtracting the third and the fourth equation (21) term by term, it results

$$(A_{31} - A_{41})\ddot{x}_1 = B_3 - B_4 \Rightarrow \ddot{x}_1 = \frac{B_3 - B_4}{A_{31} - A_{41}}. \quad (22)$$

We obtain a system of three equations with three unknowns

$$\begin{aligned} A_{11}\ddot{x}_2 + A_{13}\ddot{\theta}_3 + A_{14}\ddot{\theta}_4 &= B_1 - A_{11} \frac{B_3 - B_4}{A_{31} - A_{41}} = D_1; \\ A_{21}\ddot{x}_2 + A_{23}\ddot{\theta}_3 + A_{24}\ddot{\theta}_4 &= B_2 - A_{21} \frac{B_3 - B_4}{A_{31} - A_{41}} = D_2; \\ A_{32}\ddot{x}_2 + A_{33}\ddot{\theta}_3 + A_{34}\ddot{\theta}_4 &= B_3 - A_{31} \frac{B_3 - B_4}{A_{31} - A_{41}} = D_3. \end{aligned} \quad (23)$$

Denoting

$$\begin{aligned} \Delta = \begin{vmatrix} A_{11} & A_{13} & A_{14} \\ A_{21} & A_{23} & A_{24} \\ A_{32} & A_{33} & A_{34} \end{vmatrix}; \Delta_{x_2} &= \begin{vmatrix} D_1 & A_{13} & A_{14} \\ D_2 & A_{23} & A_{24} \\ D_3 & A_{33} & A_{34} \end{vmatrix}; \Delta_{\theta_3} = \begin{vmatrix} A_{11} & D_1 & A_{14} \\ A_{21} & D_2 & A_{24} \\ A_{32} & D_3 & A_{34} \end{vmatrix}; \\ \Delta_{\theta_4} &= \begin{vmatrix} A_{11} & A_{13} & D_1 \\ A_{21} & A_{23} & D_2 \\ A_{32} & A_{33} & D_3 \end{vmatrix}, \end{aligned} \quad (24)$$

one obtains the equilibrium conditions

$$\Delta_{x_2} = 0; \Delta_{\theta_3} = 0; \Delta_{\theta_4} = 0 \quad (25)$$

or, equivalently,

$$\begin{aligned} D_1(A_{23}A_{34} - A_{21}A_{13}) - D_2(A_{13}A_{34} - A_{21}A_{14}) + D_3(A_{13}A_{13} - A_{23}A_{14}) &= 0; \\ -D_1(A_{21}A_{34} - A_{32}A_{13}) + D_2(A_{11}A_{34} - A_{32}A_{14}) - D_3(A_{11}A_{13} - A_{21}A_{14}) &= 0; \\ D_1(A_{21}^2 - A_{23}A_{32}) - D_2(A_{11}A_{21} - A_{32}A_{13}) + D_3(A_{11}A_{32} - A_{21}A_{13}) &= 0. \end{aligned} \quad (26)$$

It results two situations: either  $D_i = 0$ ,  $i = \overline{1,3}$ , or the determinant of the system in the unknowns  $D_i$ ,  $i = \overline{1,3}$  is equal to zero. The second situation would imply that the equations (23) are not linear independent which is absurd. It remains

$$D_i = 0, i = \overline{1,3} \quad (27)$$

From the definition of the parameters  $D_i$ ,  $i = \overline{1,3}$ , it follows the equilibrium positions given by

$$x_1 = 0; x_2 = 0; \theta_3 = 0; \theta_4 = 0; \dot{x}_1 = 0; \dot{x}_2 = 0; \dot{\theta}_3 = 0; \dot{\theta}_4 = 0. \quad (28)$$

The study of the stability of this equilibrium position will be made numerically, considering a deviated position characterized by  $\xi_i$ ,  $i = \overline{1,8}$  with  $|\xi_i|$  sufficiently small.

We selected the following values:

$$\begin{aligned} m_1 &= 1000[\text{kg}]; m_2 = 26.68[\text{kg}]; m_3 = 46.06[\text{kg}]; m_4 = 5.52[\text{kg}]; L_3 = 0.427[\text{m}]; \\ L_4 &= 0.24[\text{m}]; k_{34} = 180[\text{Nm/rad}] \text{ for } \theta_4 \geq \theta_3, \text{ otherwise } k_{34} = 300[\text{Nm/rad}]; \\ k_{23} &= 350[\text{Nm/rad}] \text{ for } \theta_3 \geq 0, \text{ otherwise } k_{23} = 1000[\text{Nm/rad}]; k_{12} = 600000[\text{N/m}]; \\ k_{13} &= 600000[\text{N/m}]; k_{01} = 800000[\text{N/m}]. \end{aligned} \quad (29)$$

The step time is  $\Delta t = 0.001[\text{s}]$ , and the initial values are

$$\theta_4^0 = \frac{\pi}{360} [\text{rad}]; \dot{\theta}_3^0 = 0.001 [\text{rad/s}]. \quad (30)$$

The graphics are plotted in the next figures.

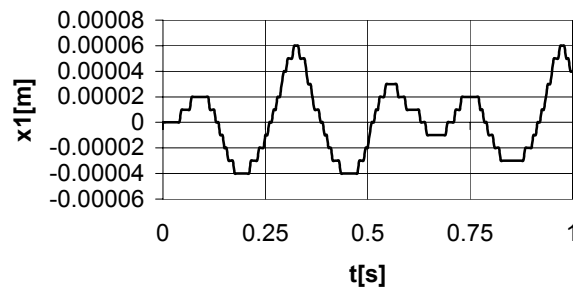


Figure 3. Time history for  $x_1$ .

It is easy to observe the quasi periodicity of the diagrams, so the equilibrium is a simply stable one.

The equilibrium becomes if  $k_{12} = 0$  for  $x_2 < 0$ .

The domain of stability increases around the equilibrium position (28) if the friction coefficient has a non-zero value.

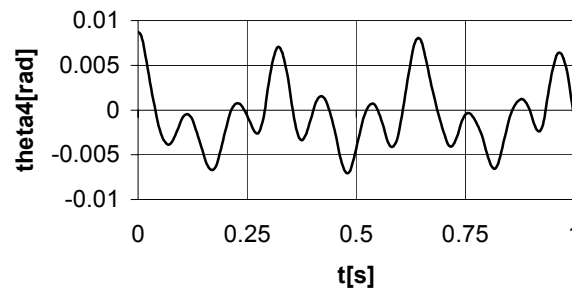


Figure 4. Time history for  $\theta_4$ .

## 5. CONCLUSIONS

In our paper we presented a four degrees of freedom model for study the motion of a human body. This model characterizes the behavior of the human body in a car. We obtained the equations of motion and we study the stability of the equilibrium position in a particular case.

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